



Introduction

TABLE 12.1

Comparison of Characteristics of Proportional Valves and Servo Valves

	Proportional valves	Servo valves
Type of Loop	Open	Closed
Feedback	No	Yes
Accuracy	Moderate error factor ≥ 3%	Extremely high error factor < 1%
Frequency response	Low: <10 Hz	Very high: 60–400 Hz
Cost	Moderate	High
Need for auxiliary electronic equipment	Moderate	Substantial
Sensitivity to contamination	Tolerant	High





Stroke-Controlled

- With a Stroke-Controlled valve, the stroke distance is proportional to the input signal.
- Both types provide an opening of the valve proportional to the magnitude of the milliamp current applied to the valve.









- **D** Proportional Directional Control Valve
- □ It has many of the same features of conventional fourway, three position DCV.
- □ Graphic symbols have orifices to indicate that the spool has been machined to allow metering of the flow.
- □ Spool in a proportional DCV can be machined with different shaped notches in the spool lands.







- □ A special spool is needed to control a cylinder with a 2:1 area ratio.
- □ Flow from the rod end during extension is half the flow to the cap end.
- □ A spool with equal triangular notches will have unequal pressure drops across the valve because of the unequal flow through equal notch areas.







- Pressure in Chamber C acts against the right end of the spool causing it to move to the left against the main spool spring.
- □ The main spool spring is a push-pull spring.
- Pressure in Chamber D acts on the end of the piston that seals the left end of the spring cavity.
- □ Higher the pilot pressure, greater the spool displacement.



- □ The programmable orifice is set to create a ΔP , which, when added to other ΔP s in the circuit, creates a pressure that opens the relief valve.
- □ Flow to the actuator is controlled by the dumping part of the pump flow across the relief valve.
- A proportional DCV controls flow by converting hydraulic energy to heat energy, thus need to provide for cooling of fluid.







- □ At 12 Hz, the phase lag for the +/- 25 % signal is approximately 45° , meaning that the output lags the input by 45° .
- □ Phase lag at 9 Hz is also approximately 45° for the +/- 50% signal.
- □ As signal frequency increases, ability of valve to "keep up" decreases and control is lost. If frequency increases to phase angle 180° meaning the system is unstable.





Analysis of Proportional Directional Control Valve The Orifice equation applies for both sides of the valve. $Q_1 = CA_1\sqrt{\Delta P_1}$ $Q_2 = CA_2\sqrt{\Delta P_2}$ Where $Q_1 =$ flow into cap end of cylinder (in³/s) $Q_2 =$ flow out of rod end of cylinder (in³/s) $A_1 =$ area of orifice between Port P and Port A(in²) $A_2 =$ area of orifice between Port B and Port T(in²) C = orifice coefficient (in².s⁻¹.lb_f^{-0.5}) $\Delta P_1 =$ pressure drop between Ports P and A (psi) $\Delta P_2 =$ pressure drop between Ports B and T (psi)



Analysis of Proportional Directional Control Valve Flow out rod end for each inch of movement is half the flow into the cap end. $A_2 = A_1/2$ Substituting this value, $Q_1 = CA_1\sqrt{\Delta P_1}$ $Q_2 = Q_1/2 = C(A_1/2)\sqrt{\Delta P_2}$ (or) $Q_1 = CA_1\sqrt{\Delta P_2}$ (only if $\Delta P_1 = \Delta P_2$)





- \square F_L is negative, since the load is overrunning, i.e., it is acting in the direction of cylinder movement.
- \Box Solving for P_r ,

$$P_r = (P_c A_c + F_{L-} F_f) / A_r$$

Pressure drop across the Port P to Port A orifice in the proportional valve is

$$\Delta P_1 = P_S - P_C$$

Analysis of Proportional Directional Control Valve

Neglecting pressure drop between the proportional valve outlet and the reservoir, P_o=0, the pressure drop from Port A to T is

$$\Delta P_2 = \Pr - P_o = \Pr$$

□ With a 1:1 area ratio value, $A_1 = A_2 = A$, and the orifice equations become

$$Q_1 = CA\sqrt{\Delta P_1} \qquad \qquad Q_2 = CA\sqrt{\Delta P_2}$$

Solving for CA and equating the two expressions,

$$Q_1/\sqrt{\Delta P_1} = Q_2/\sqrt{\Delta P_2}$$
(or)
$$Q_1/Q_2 = \sqrt{\Delta P_1}/\sqrt{\Delta P_2}$$
Squaring both sides,

$$Q_1^2/Q_2^2 = \Delta P_1/\Delta P_2$$
(or)
$$\Delta P_2 = \Delta P_1Q_2^2/Q_1^2$$

Analysis of Proportional Directional Control Valve

 $\square Substituting for \Delta P_1 and \Delta P_2,$

$$\mathbf{Pr} = (Ps - Pc)Q2^2 / Q1^2$$

 Equating, and solving for pressure at the cap end of the cylinder.

$$\frac{(P_cA_c + F_L - F_f)}{A_r} = (P_s - P_c)Q_2^2/Q_1^2$$
$$P_c = \frac{P_s(Q_2^2/Q_1^2) - (F_L - F_f)/A_r}{(A_c/A_r) + (Q_2^2/Q_1^2)}$$





- □ This equation is the same as 12.6 except for the change in sign for F_L .
- □ Since the value has a 1:1 area ratio,

$$\Delta P_2 = \Delta P_1 Q_2^2 / Q_1^2$$

(or)

$$\Delta P_1 = \Delta P_2 Q_1^2 / Q_2^2 \dots (Eq. 12.20)$$

Analysis of Proportional Directional Control Valve

□ As previously defined,

$$\Delta P_1 = P_S - P_C$$

$$\Delta P_2 = \Pr$$
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 \square Substituting into Eq. 12.20 and solving for P_c,

$$P_c = P_s - \Pr Q_1^2 / Q_2^2$$
.....(Eq.12.21)

 \square Equating Eq. 12.19 and Eq.12.21 and solving for P_r,

$$P_{r} = \frac{P_{s} - (F_{f} + F_{L}) / A_{c}}{\frac{A_{r}}{A_{c}} + \frac{Q_{1}^{2}}{Q_{2}^{2}}}$$

